NATIONAL ADVISORY COMMITTEE FOR AERONAUTICS

WARTIME REPORT

ORIGINALLY ISSUED

October 1945 as Memorandum Report E5J03

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WITH A NEOPRENE-COATED VALVE GRID

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WASHINGTON

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NATIONAL ADVISORY COMMITTEE FOR AERONAUTICS

MEMORANDUM REPORT

for the

Air Technical Services Command, Army Air Forces

and the

Bureau of Aeronautics, Navy Department

ENDURANCE TESTS OF A 22-INCH-DIAMETER PULSE-JET ENGINE

WITH A NEOPRENE-COATED VALVE GRID

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SUMMARY

Thrust-stand tests were conducted at high thrust outputs to determine the operating life of a 22-inch-diameter pulse-jet engine equipped with a neoprene-coated valve grid. The results of the endurance tests show that through the use of the neoprene-coated grid the operating life of the pulse-jet engine, as limited by valve deterioration, was extended to more than 164 minutes, as compared with 30 minutes for the standard uncoated grid. The average jet thrust (not deducting the momentum drag of the entering air) developed by the engine was 855 pounds at a simulated ram pressure of 58 inches of water and a fuel flow of 2800 pounds per hour; no decrease in thrust was obtained during the entire 164 minutes of operation. This jet-thrust value represents a slight reduction in performance from the average 890 pounds of thrust obtained with the standard valve grid under similar operation conditions.

INTRODUCTION

At the request of the Air Technical Service Command, Army Air Forces, and of the Bureau of Aeronautics, Navy Department, an investigation is being conducted at the NACA Cleveland laboratory to improve the performance and extend the operating life of the pulse-jet engine. As a part of this investigation, thrust-stand tests were conducted to determine the sea-level performance of a

22-inch-diameter pulse-jet engine at simulated rem pressures of 0, 18, 40, and 58 inches of water over the entire fuel-flow operating range of the jet engine (reference 1).

The results of reference 1 and of tests conducted at Wright Field (reference 2) indicate that the operating life of the pulse-jet engine, as limited by valve deterioration caused by the repeated impact forces imposed on the valves in closing during operation, is approximately 30 minutes, after which a rapid reduction in engine-thrust output is obtained.

Attempts to increase valve life through change of valve material and thickness have been unsuccessful, oral reports from Wright Field, obtained subsequent to the tests reported perein indicate, however, that the time of satisfactory operation can be increased to about 1 hour by careful selection, honing, and finishing of the valves.

The method of reducing valve deterioration investigated in the present tests consists in diminishing the valve shock forces by cushioning the valves through use of an energy absorbent material on the valve seats. The entire grid of face of a production flapper-valve assembly was coated with a thin layer of neoprene by means of a process developed by the B. F. Goodrich Company of Akron, Ohio. Thrust-stand endurance tests of this modified valve assembly installed in a pulse-jet engine was conducted at high thrust outputs during July 1945 and the results are presented herein.

APPARATUS AND METHODS

The 22-inch-diameter (maximum) pulse-jet engine used in the tests is described in references 1 and 2. The principal dimensions of the engine sholl and the positions of the valve-grid assembly and the venturi are shown in figure 1. The thrust test stand and the method of simulating ram pressure and other installation and instrumentation details are essentially the same as those described in reference 1. In order to obtain a better indication of the valve-grid operating temperatures, the thermocouples previously installed on the upstream face of the grid (see reference 1) were transferred to the downstream face. The process used by the B. F. Goodrich Company to neoprene-coat the valve grid is outlined as follows:

(a) The grid sections are degreased, painted with a primer, brushed with neoprene cement, dipped in a coagulant and then in neoprene latex, washed with water, and drained with the trailing edge up.

channels is probably to improve mixing between the combustion air and the fuel. A spark plug installed on the combustion-chamber shell in the position indicated in figure 1 is used for starting the engine.

Figures 2 and 3 are photographs of the downstream and upstream faces of the flapper-valve grid assembly and show the nine centrifugal-spray fuel-injection nozzles, the three compressed-air jets, the fuel-distribution manifold, and the compressed-air lines. The compressed-air jets, which are incorporated for static starting, were not used in the tests.

Resonant operation of the engine is as follows: The flapper valves open and admit air into the combustion chamber. The fuel spray mixes with the incoming air and forms a combustible mixture, which is ignited by the residual exhaust gases of the previous explosion. The pressure rise resulting from the explosive combustion causes the flapper valves to snap shut against the grid and the burned products of combustion are discharged rearwardly through the toil pipe, thus providing a thrust impulse on the engine in a forward direction. The outward rush of gases from the tall pipe reduces the pressure within the combustion chamber, which causes the flapper valves to reopen and admit a new charge of air. The cycle than reports itself at a frequency governed by the resemant frequency of the engine tube.

Thrust stand and thrust-magsuring system. - Figure 4 is a schematic diagram of the test sutup showing the mounting frame, the thrust plitform and thrust-marsuring linkago, and the general ducting system for supplying combustion air to the engine. The shrouds and ducting of the cooling-air system and other details of the test installation may be seen in the photograph of figure 5. The front (upstream) support for the engine consists of an adjustable yoke having floxible rubbor-mounted bearings at the two support points for cushioning the violently fluctuating thrust forcer obtained during operation. The rear (downstroam) support is fixed to the frame with a simple sliding bolt and slot arrangement to provide for the expansion of the shell whon heated during normal operation. After initial tests, the extreme and of the tail pipe was clamped to the frame by means of metal straps to prevent it from whipping and onegging during operation. The thrust platform, upon which the mounting frame is belted, is supported by ball and rolls bearing linkages above a bedplate securely anchored to an isolated concrete block set in the ground.

The thrust platform, the support linkages, and the bedplate form the sides of a pin-jointed parallologram. The rear support linkages and a thrust arm are keyed to a common shaft to form a bell-crank arrangement with a 10:1 leverage ratio. The forces on the engine are transmitted from the thrust platform, through the bell crank, to the piston of a hydraulic piston-cylinder assembly. Korosene is

- F predicted sea-level flight thrust, pounds
- Vo free-stream flight velocity corresponding to the simulated ram pressure at which F₁ is obtained, feet per second

RESULTS AND DISCUSSION

A summary of the data obtained during the first 43.6 minutes of testing at variable operating conditions is presented in table I. A similar summary of the data subsequently obtained at constant operating conditions (simulated ram pressure, 58 in. water; fuel flow, 2800 lb/hr) is presented in table II. The run numbers listed in the tables indicate periods of continuous operation of the engine. Values of both the jet thrust Fj (see equation (1)) and the corresponding predicted sea-level flight thrust F (see equation (2)) are included in the tables.

The jet thrust and the combustion-air weight flow for the constant operating conditions (data from table II) are plotted against total operating time in figures 2 and 3, respectively. Inspection of these figures indicates that the engine apparetnly operates between a low-power and a high-power level at irregular intervals; during the tests this change in power level was accompanied by a very noticeable change in noise intensity and in the amplitude of the induced vibrationa. The explanation for the sporadic variation in power level is not known. The average jet thrust obtained during the constant operating condition (fig. 2) is 855 pounds and the average combustion-air flow (fig. 3) is 10.75 pounds per second. During the tests of reference 1 a standard uncoated valve-grid assembly operating at the same ram pressure and fuel-flow conditions developed an average jet thrust of approximately 890 pounds at a combustion-air flow rate of 11 pounds per second. This result would indicate a slightly adverse effect of the neoprene-coated valve-grid assembly on the performance of the pulse-jet engine, which is attributed to the reduction in free-flow area of the grid caused by the neoprene coating.

The jet thrust and the combustion-air weight flow show no tendency to drop (figs. 2 and 3) even after 163.6 minutes of operation, indicating that the valve deterioration at this point is of insufficient magnitude to affect performance. The condition of the flapper-valve grid assembly after a total operating time of 51.6 minutes and 163.6 minutes is shown in figures 4 and 5, respectively. After 51.6 minutes of operation no deterioration of the valves was visible except for slight discolorations, which apparently were not harmful. After 163.6 minutes of operation, one valve was completely broken off near the rivet holes, evidently due to fatigue in flexure, and three other valves were beginning to split and fray near the trailing edges. Although the valve assembly could possibly have been operated for additional time without appreciable loss in performance, 163.6 minutes is taken as a conservative estimate of the life of the neoprene-coated flapper-valve grid assembly for the operating condition at high thrust output. This value of valve life represents an appreciable increase over the 30-minute life of a standard unit. Observations of the temperature of the downstream face (flame side) of the valve grid during the last part of the tests indicated a maximum grid temperature of 280° F, which is well below the 380° F safe limit specified for the neoprene coating.

Inspection of the grid at intervals during the course of the tests revealed a slightly adhesive or tacky condition of the rubber coating, which caused the valves to stick to the grid. Although this condition apparently did not affect the performance of the engine, it might possibly have an adverse offect on static starting of the engine. Representatives of the B. F. Goodrich Company attribute the tacky condition to the presence of excess quantities of the coagulant used in the coating process and not to the characteristics of the neoprone itself, which tends to harden with prolonged exposure to heat. They believe that careful washing of the coating prior to the drying and curing phases of the process will give the desired results.

SUMMARY OF RESULTS

The results of thrust-stand tests at high thrust outputs of a 22-inch-diameter pulse-jet engine equipped with a neoprene-coated valve-grid indicate that:

- 1. The operating life of the pulse-jet engine, as limited by valve deterioration, was longer than 164 minutes as compared with 30 minutes for the standard valve assembly.
- 2. The average jet thrust developed by the engine was 855 pounds at simulated ram pressure of 58 inches of water and fuel flow of 2800 pounds per hour and did not depreciate during the 164 minutes of operation. This value of thrust represents a 4-percent reduction in performance from the average 890 pounds of thrust obtained with the standard valve grid under similar operating conditions.

Aircraft Engine Research Laboratory,
National Advisory Committee for Aeronautics,
Cleveland, Ohio, October 3, 1945.

REFERENCES

- 1. Magnaniello, Eugene J., Valerino, Michael F., and Essig, Robert H.: Sea-Level Performance Tests of a 22-Inch-Diameter Pulse-Jet Engine at Various Simulated Ram Pressures. NACA MR No. E5J02, 1945.
- Bogert, R. C.: Life Test of Ford MX-544 Intermittent Jet Engine. Memo. Rep. Ser. No. TSEPL-5-673-56, Eng. Div., Army Air Forces, Dec. 13, 1944.

TABLE I - SUMMARY OF TEST DATA AT VARIABLE OPERATING TEST CONDITIONS

Run	Total operating time of flapper valve at end of run (min)	Simulated ram pres- sure (in. water)	Fuel flow (lb/hr)	Fuel- nozzle pressure (lb/sq in. gage)	Combustion- air weight flow (lb/hr)	Combustion- air temper- ature (°F)	Jet thrust (1b)	Predicted sca-level flight thrust (lb)
ı	3. i	58.5 58.7 59.0 58 .6	3400 3200 2800 2400	58 53 41 31	39,960 40,320 37,800 36,360	80 77 78 79	842 883 866 805	668 707 700 646
2	6.0	59.1 58.5 55.5 59.3	3400 3200 2800 2400	59 53 40 32	39,600 38,880 37,440 36,000	128 128 131 133	753 858 786 692	579 688 627 533
3	9.0	38.4 36,4 39.9 40.8	3000 2800 2400 2200	37 33 24 20	35,640 32,400 30,960 30,240	139 137 138 137	728 699 599 551	6 0 2 58 8 48 8 44 0
4	12.0	37.6 38.0 40.0 40.2	3000 2800 2400 2200	46 40 30 25	36,360 35,280 32,040 31,320	88 86 87 86	758 731 699 652	631 607 583 538
5	13.6	58.8 58.9	3400 2800	58 5 I	40,320 37,800	86 86	867 847	69 I 68 2
6	16.4	38.8 37.8 37.8 40.1	3000 2800 2400 2200	40 31 27	33,840 35,280 31,320 30,960	135 135 136 136	709 738 650 639	589 614 540 528
7	17.3	19.1	2400		30,960	135	531	454
8	19.9	57.4 57.6	3200 2800		39,600 37,800	85 84	794 843	623 679
9	22.4	59.0 58.0 59.1 58.7	2400 3200 2800 2400	31 51 40 31	36,000 38,160 39,960 35,640	85 130 132 133	743 774 800 745	585 608 624 589
10	25.0	38.7 38.8 38.9	2800 2400 2200	37 30 25	33,480 34,920 30,960	134 134 135	725 661 640	60 6 537 530
11	27.5	37.€ 36.7 40.3	2800 2400 2200	39 30 25	35,280 32,040 31,320	94 93 93	76 723 646	638 612 532
12	30.1	15.8 20.4 20.5	2400 2000 1600	25 21 	28,080 26,640 24,120	79 79 79	644 558 363	58 I 490 30 I
13	32.7	18.7 19.3 19.8	2400 2000 1600	25 21	28,080 27,360 24,120	126 128 128	586 509 429	513 440 368
14	35.4	38.5 38.7 37.6	3000 2800 2 4 00	43 39 31	33,480 34,200 31,680	134 135 136	703 737 6 96	584 616 595
15	38.2	37.5 37.1 37.4	3000 2800 2400	42 40 31	35,640 34,920 32,040	9 I 89 89	773 764 702	649 642 590
16	40.9	57.9 57.2 60.3	3200 2800 2400	50 41	39,960 37,440 36,000	85 84 83	781 821 749	608 659 589
17	43,6	58.4 57.4 50.3	3200 2800 2400	50 41 31	38,520 37,800 35,640	135 136 137	788 784 694	610 620 536

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TABLE II - SUMMARY OF TEST DATA AT CONSTANT
OPERATING TEST CONDITIONS

[Nominal simulated ram pressure, 58 in. water; fuel flow, 2800 lb/hr; fuel-nozzle pressure, 41 lb/sq in. gage; combustion-air temperature, 94° F]

Run	Total operating time of flapper valve at end of run (min)	Simulated ram pres- sure (in. water)	Combustion- air weight flow (lb/hr)	Jet thrust (1b)	Predicted sea-level flight thrust (lb)
18	46.8	58.4	36,720	766	605
19	49.9	59 .2	36,720	766	605
20	51.6	59.0	36,360	702	540
21	59.1	58.0 56.8 57.3	39,960 37,800 37,440	876 814 838	702 651 676
22	74.1	60.0 57.9 57.5 57.6 58.6 57.8 58.5 57.2 57.8 57.2 56.9 58.1 58.0	38,520 38,520 38,160 38,160 39,600 39,600 37,080 37,800 38,160 37,440 37,800 37,440 38,520	867 851 868 837 863 812 794 860 879 897 868 853 820 838	696 683 703 672 696 640 621 698 716 732 706 690 657 671
23	87.0	57.0 57.6 57.2 57.7 58.3 57.9 58.5	37,440 37,800 38,520 39,600 38,160 39,600 37,800	843 853 875 867 894 944 873	68 2 689 708 696 728 772 708

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TABLE II - SUMMARY OF TEST DATA AT CONSTANT
OPERATING TEST CONDITIONS - Continued

Run	Total operating time of flapper valve at end of run (min)	Simulated rem pres- sure (in. water)	Combustion- air weight flow (lb/hr)	Jet thrust (1b)	Predicted sea-level flight thrust (lb)
24	112.1	58.2 58.1 58.4 57.4 58.4 57.2 57.3 55.3 58.0 58.0 56.8	39,960 37,800 37,800 39,240 37,440 37,800 37,800 37,800 37,440 38,520 37,440	841 872 857 910 820 861 840 907 845 843 865	667 707 692 740 657 698 677 746 682 675 704
25	142.1	58.0 59.2 60.1 57.1 59.0 57.3 57.0 56.3 57.7 57.0 58.0 58.0 58.0 58.0 59.0	39,600 39,960 39,960 39,960 39,960 38,880 38,880 38,520 37,800 39,600 37,440 39,240 39,960 37,800 39,600	839 851 8 59 914 926 788 861 788 872 890 862 897 839 886 865 860 900	667 675 682 741 750 615 693 622 698 721 723 696 725 676 716 691 696 727

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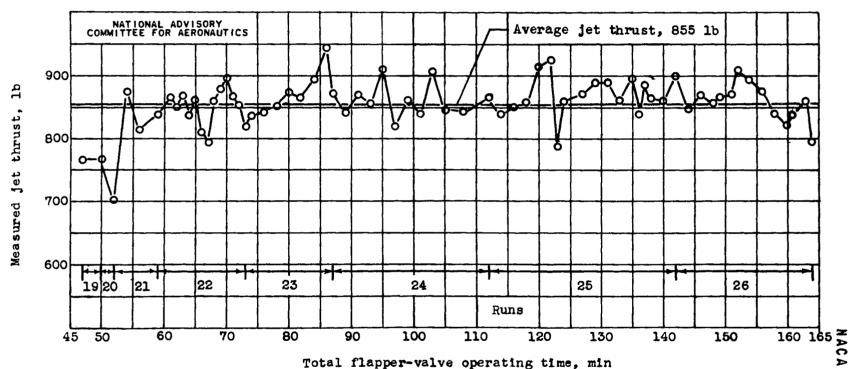


Figure 2.- Variation of measured jet thrust with flapper-valve operating time. Average simulated ram pressure, 58 inches of water; fuel flow, 2800 pounds per hour.

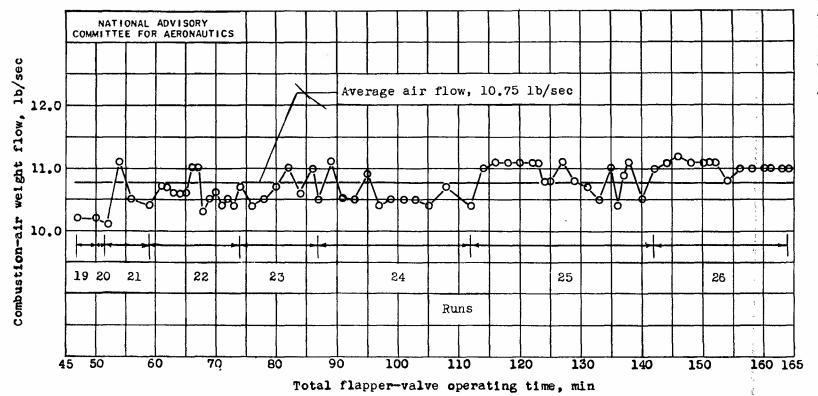


Figure 3.- Variation of combustion-air weight flow with flapper-valve operating time. Average simulated ram pressure, 58 inches of water; fuel flow, 2800 pounds per hour.

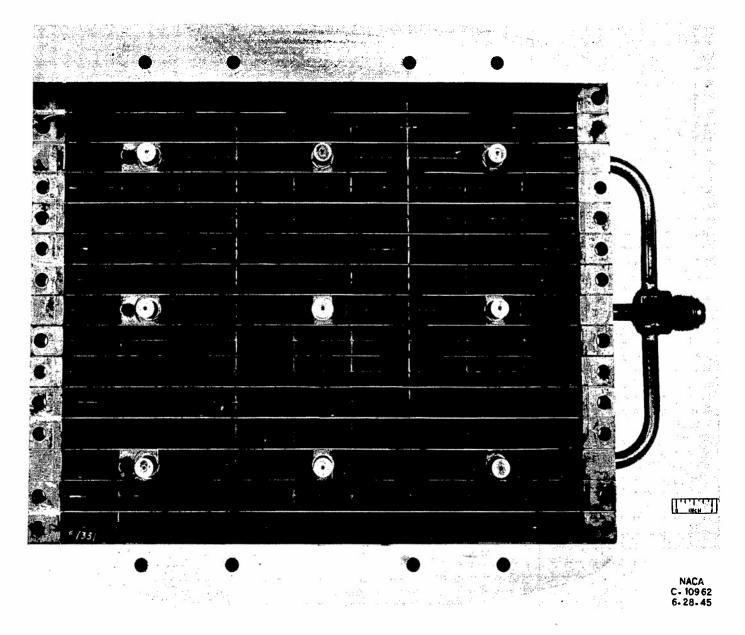


Figure 4. - Neoprene-coated flapper-valve grid assembly after 51.6 minutes (after run 20) operating time.

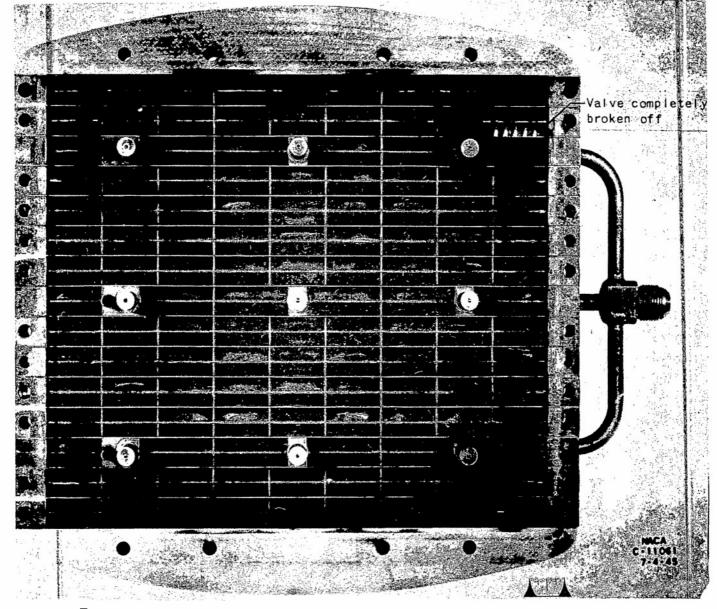


Figure 5. - Neoprene-coated flapper-valve grid assembly after 163.6 minutes operating time.